

13.021 – Marine Hydrodynamics

Lecture 18

4.7 – Turbulent Flow – Reynolds Stress

$$u = \bar{u} + u'$$

By definition: $\bar{u'} = \overline{u - \bar{u}} = \bar{u} - \bar{u} = 0$, also $\frac{\partial}{\partial x} \bar{u} = \frac{\partial \bar{u}}{\partial x}$ etc

Substitute into governing equations and take $(\bar{\quad})$:

$$\text{continuity: } 0 = \frac{\partial u_i}{\partial x_i} = \frac{\partial \bar{u}_i}{\partial x_i} + \underbrace{\frac{\partial u'_i}{\partial x_i}}_0 \quad \therefore \frac{\partial \bar{u}_i}{\partial x_i} = 0$$

$$\text{but } \frac{\partial u_i}{\partial x_i} = 0 = \underbrace{\frac{\partial \bar{u}_i}{\partial x_i}}_{0 \text{ just shown}} + \frac{\partial u'_i}{\partial x_i} \quad \therefore \frac{\partial u'_i}{\partial x_i} = 0$$

$$\text{Momentum Equation: } \frac{\partial u_i}{\partial t} + u_j \frac{\partial u_i}{\partial x_j} = \frac{1}{\rho} \frac{\partial \tau_{ij}}{\partial x_j} = -\frac{1}{\rho} \frac{\partial p}{\partial x_i} + \nu \nabla^2 u_i$$

$$u_i = \bar{u}_i + u'_i \text{ \& } (\bar{\quad}):$$

$$\frac{\partial u_i}{\partial t} = \frac{\partial \bar{u}_i}{\partial t} + \underbrace{\frac{\partial u'_i}{\partial t}}_0; \text{ similarly } \begin{cases} \overline{\nu \nabla^2 u_i} = \nu \nabla^2 \bar{u}_i \\ \frac{\partial \bar{p}}{\partial x_i} = \frac{\partial}{\partial x_i} (\bar{p} + p') = \frac{\partial \bar{p}}{\partial x_i} \text{ etc} \end{cases}$$

$$\overline{u_j \frac{\partial u_i}{\partial x_j}} = \overline{(\bar{u}_j + u'_j) \frac{\partial}{\partial x_j} (\bar{u}_i + u'_i)} = \bar{u}_j \frac{\partial \bar{u}_i}{\partial x_j} + \underbrace{\bar{u'_j} \frac{\partial \bar{u}_i}{\partial x_j}}_0 + \underbrace{\bar{u}_j \frac{\partial u'_i}{\partial x_j}}_0 + \underbrace{\bar{u'_j \frac{\partial u'_i}{\partial x_j}}}_0$$

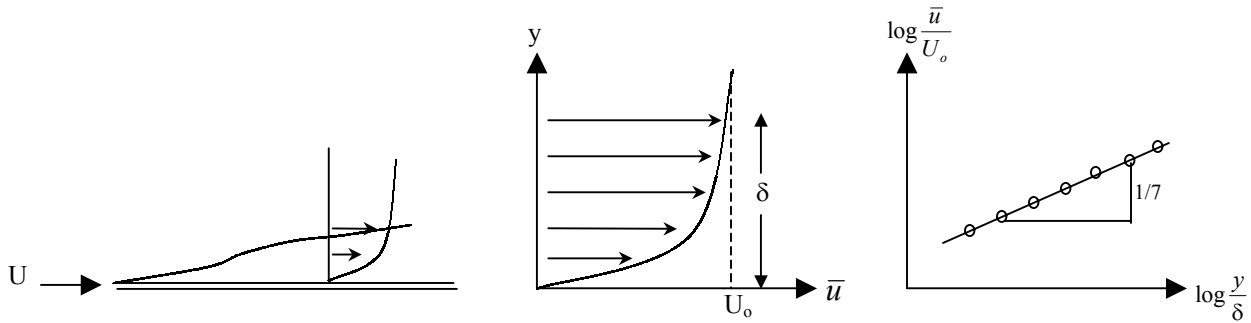
$$\text{From continuity: } \overline{u'_j \frac{\partial}{\partial x_j} u'_i} = \frac{\partial}{\partial x_j} \overline{u'_j u'_i} - \underbrace{u'_i \frac{\partial u'_j}{\partial x_j}}_0 \text{ from continuity}$$

$$\text{Finally: } \frac{\partial \bar{u}_i}{\partial t} + \bar{u}_j \frac{\partial \bar{u}_i}{\partial x_j} = -\frac{1}{\rho} \frac{\partial \bar{p}}{\partial x_i} + \nu \nabla^2 \bar{u}_i - \underbrace{\frac{\partial}{\partial x_j} \overline{u'_i u'_j}}_{\frac{1}{\rho} \frac{\partial}{\partial x_j} \bar{\tau}_{ij}}$$

$$\frac{\partial \bar{u}_i}{\partial t} + \bar{u}_j \frac{\partial \bar{u}_i}{\partial x_j} = \frac{1}{\rho} \frac{\partial}{\partial x_j} \left[\bar{\tau}_{ij} - \rho \overline{u'_i u'_j} \right] \quad \text{Reynolds averaged N-S eqts. (RANS)}$$

$$\text{Reynolds stress } \tau_{Rij} = -\rho \overline{u'_i u'_j}$$

4.8 – Turbulent Boundary Layer over a Smooth Flat Plate



(1/7)th power velocity profile law: $\frac{\bar{u}}{U_o} = \left(\frac{y}{\delta}\right)^{\frac{1}{7}}$ (1)

where $\delta = \delta(x)$ to be determined.

$$(1) \rightarrow \delta^* = \frac{\delta}{8}$$

$$\theta = \frac{7}{72} \delta \cong 0.0972 \delta$$

$$\frac{\tau_o}{\rho U_o^2} = 0.0227 \left(\frac{U_o \delta}{\nu} \right)^{-\frac{1}{4}} \quad \leftarrow \text{Using another empirical formula for friction (Blasius law of friction) for pipes}$$

Von Karman's momentum integral equation:

$$\frac{\tau_o}{\rho U_o^2} = \frac{d}{dx}(\theta) \rightarrow 0.0227 \left(\frac{U_o \delta}{\nu} \right)^{-\frac{1}{4}} = \frac{7}{72} \frac{d\delta}{dx} \leftarrow \text{ODE for } \delta$$

$$\frac{\delta}{x} \cong 0.373 R_x^{-\frac{1}{5}}$$

$$\delta(x) \cong 0.373 x \left(\frac{U_o x}{\nu} \right)^{-\frac{1}{5}}$$

For $\delta(0) = 0$ and assuming turbulent boundary layer at $x = 0$, i.e. tripped at $x = 0$ or $R_x \gg 1$

$$\Rightarrow \frac{\delta}{x} \cong 0.373 R_x^{-\frac{1}{5}}$$

$$\delta(x) \sim \sqrt{x} \quad \text{laminar}$$

$$\delta(x) \sim x^{4/5} \quad \text{turbulent (grows much faster)}$$

| | Laminar | Turbulent |
|---------|--|--|
| Blasius | $\delta^* \sim 1.72 \sqrt{\frac{\nu}{U_o}} \sqrt{x}$ | $\delta^* \sim 0.047 \left(\frac{\nu}{U_o} \right)^{1/5} x^{4/5} \dots\dots \left(\frac{1}{7} \right)^{\text{th}} \text{ power law}$ |

$$D = 0.036 (\rho U_o^2) BL R_L^{-1/5}$$

$$C_f = \frac{D}{\frac{1}{2} \rho U_o^2 BL} = 0.073 R_L^{-1/5} \quad \text{for } R_L > 5 \times 10^5$$

Logarithmic Velocity Profile Law

$$\frac{0.242}{\sqrt{C_f}} = \log_{10} (R_L C_f) \leftarrow \text{Schoenherr's formula}$$